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Experimental and Numerical Characterization of CaPaMan 2bis Operation

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ABSTRACT

In this paper, the operation performance of CaPaMan 2bis (Cassino Parallel Manipulator 2 bis) was investigated through lab experimental tests and computer simulations. Milli-CaTraSys (Milli-Cassino Tracking System) was implemented to determine displacements and orientation variations of CaPaMan 2bis end-effector during experimental tests. A 3D (three dimension) virtual model was built in ADAMS environment to simulate the operation behavior for different prescribed motions. Several prescribed motions have been simulated and tested under different conditions in order to characterize the system behavior. In particular, the kinematic characteristics were obtained from both, experimental tests and numerical simulations. Finally, experiment results and simulation computations were compared for purpose of performance evaluation and design characterization of the parallel manipulator structure and its prototype.

Keywords: Experimental mechanics, parallel manipulators, simulation, kinematics

RESUMEN

En este trabajo se investiga el desempeño de la operación del manipulador paralelo CaPaMan 2bis a través de pruebas experimentales y de simulaciones numéricas. El sistema Milli-CaTraSys se implementó para determinar la variación de los desplazamientos y la orientación del órgano terminal de CaPaMan 2bis durante las pruebas experimentales. Un modelo 3D desarrollado en ADAMS fue utilizado para simular el comportamiento operacional para diferentes movimientos prescritos. Tales movimientos han sido simulados y verificados bajo diferentes condiciones con el fin de caracterizar el comportamiento del sistema. En particular, se obtienen características cinemáticas de las pruebas experimentales y de las simulaciones numéricas. Finalmente, se comparan los resultados experimentales con los cálculos de simulación con el propósito de evaluar el desempeño y la caracterización del diseño del manipulador paralelo y su prototipo.

1. Introduction

Parallel manipulators are well understood for higher payload capability, stiffness, accuracy and dynamic performance in contrast to traditional serial manipulators [1, 2]. A parallel manipulator has the main property of having its gripper connected to the base through several serial (and/or parallel) kinematic chains. Nowadays, some commercial parallel robots are already available in the market for different applications such as packaging, materials handling and machining [3]-[5]. Nevertheless, the parallel robots field has attracted research interests for the last

two decades and many prototypes have been built in laboratories for academic research. As a consequence, the number of works that deal with open problems of this mechanism type has also grown. For example, an extensive survey on commercial applications and state of the art of parallel robots can be found in the web page [6]. In fact, problems of parallel robots are still open for efficient exploitation of their inherent advantages, mainly small workspace, and complexity of direct kinematic equations or poor dexterity [7]. For this reason, it can be interesting to study mechanisms with a minor mobility that are able to perform elementary motions, namely translation, rotation

or planar movements. In addition, many applications can be achieved with less than six degree of freedom (DOF). Indeed, a three DOF mechanism could be enough, thus reducing the total cost and operations for end-effector positioning.

Αt LARM: Laboratory of Robotics and Mechatronics, University of Cassino, several prototypes of parallel manipulators have been built with low-cost and easy-operation characteristics. Research activities have been carried out on both theoretical study and application aspects. A three DOF spatial parallel manipulator named as CaPaMan has been proposed with peculiar mechanism architecture [8]. This parallel manipulator has been studied to analyze its architecture in terms of workspace, kinematic formulation, optimal design, stiffness properties and dynamic performance, [2] [9]-[12]. In addition, the platform of CaPaMan has been used as a motion generator for earthquake simulation [13]. Later, a new version called CaPaMan 2 has been conceived and built at LARM. Each prismatic joint on the coupler for each CaPaMan leg has been substituted by a revolute joint. Afterwards, CaPaMan 2 has been aimed for drilling applications, and then stiffness and accuracy have been improved by paying special attention to mechanical design. CaPaMan 2bis has been the result of these enhancements, where the modified shape of the movable plate has been designed to properly lock a commercially available drilling tool. A prototype of CaPaMan 2bis built at LARM is shown in Figure 1(a). It has been implemented as a part of a hybrid robotic architecture for surgical tasks in the work [14], as well as a trunk module in a humanoid robot design named as CALUMA (CAssino Low-cost hUMAnoid robot) [15]. Previous works were developed to analyze the CaPaMan 2bis prototype in terms of stiffness and basic kinematics [16, 17]. Since the solution of direct kinematic problem for CaPaMan 2bis can be complex, the system behavior can be

conveniently simulated and experimentally measured.

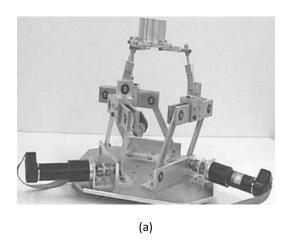
In this paper, operational performances of 2bis investigated both CaPaMan are experimental tests and simulation. Milli-CaTrays, a measuring system built at LARM was implemented to measure the position and orientation of the end-effector on the platform of CaPaMan 2bis. Several motions with different conditions were prescribed for characterizing the system behavior. Simulation was carried out in ADAMS environment in order to investigate the feasibility of this parallel Experiment results and simulation computations were compared for the purpose of operation performance evaluation and design characterization of the architecture design and its prototype.

2. Parallel manipulator design

A kinematic scheme of CaPaMan 2bis is shown in Figure 1(b). It is composed of a movable platform (MP) connected to a fixed base (FP) through three leg mechanisms. Each leg mechanism is composed of an articulated parallelogram (AP) whose coupler carries a revolute joint (RJ), a connecting bar (CB) that transmits the motion from AP to MP through RJ, and a spherical joint (BJ), which is installed on MP at point J. Revolute joint RJ installed on the coupler of AP has the rotation axis coinciding with the parallelogram plane. Each leg mechanism is rotated $2\pi/3$ with respect to the neighboring one so that the leg planes lie along two vertices of an equilateral triangle, giving symmetry properties to the mechanism. Design parameters of the mechanism are shown in Figure 1(b). They can be identified for the k-th leg mechanism (k=1, 2, 3) as: a_k is the length of the frame link; b_k is the length of the input crank; c_k is the length of the coupler link; d_k is the length of the follower crank and h_k is the length of the connecting bar. The size of movable platform MP and fixed base FP are given by distances r_p and r_f , as shown in Figure 1(b). Points

H and O are the center points of MP and FP, respectively. Points O_k are the middle point of frame link a_k , J_k are the connecting points between

the *k*-th leg mechanism and platform MP. The design parameters of the CaPaMan 2bis are listed in Table I.



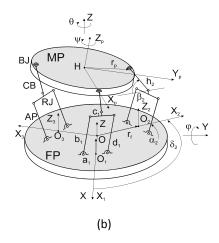


Figure 1. Cassino Parallel Manipulator 2bis: (a) A prototype built at LARM; (b) A kinematic scheme of CaPaMan 2bis.

$a_k = c_k \text{ [mm]}$	$b_k = d_k \text{ [mm]}$	r_p [mm] r_f [mm		h_k [mm]	$I_{cxy}=I_{cxy}=I_{cyz}$ [kg.mm ²]		
100	100	65	65	50	0		
α_k [deg]	β_k [deg]	<i>m</i> [kg]	τ [N/m]	I_{czz} [kg.mm ²]	$I_{cxx}=I_{cxy}$ [kg.mm ²]		
45 to 135	30 to 120	2.3	2	24600	12400		

Table 1. Mechanical design parameters and inertial properties of CaPaMan 2bis.

An inertial frame O-XYZ has been assumed to be fixed to base FP. A moving frame $HX_PY_PZ_P$ has been attached to platform MP. O-XYZ has been fixed with Z axis orthogonal to the FP plane and X axis as coincide with the line joining O to O_1 . Moving frame $HX_PY_PZ_P$ has been fixed to platform MP with Z_P orthogonal to the MP plane and X_P axis as coincident to the line joining H to J and Y_P to give a Cartesian frame. Angle δ_k is the structure rotation angle between OX_1 and OX_k as well as between HJ_1 and HJ_k . They are equal to δ_1 = 0, δ_2 =2 $\pi/3$ and δ_3 = 4 $\pi/3$.

The position of H in the center of movable platform MP is represented by coordinates x, y and z. The orientation of MP with respect to fixed base FP can be represented with the three Euler angles θ , ϕ and ψ as shown in Fig. 1(b), where θ is a rotation about the Z axis; ϕ is a tilting rotation about the Y_P axis, which is the Y axis after a θ rotation; and ψ is a rotation about the Z_P axis. For each leg mechanism, the kinematic variables are input crank angles α_k and passive revolute angle β_k , which is between the connecting bar and axis Xk. Therefore, each leg mechanism has two

degrees of freedom. Thus, three legs have six degrees of mobility and the movable platform has additionally six degrees of mobility. But since each spherical joint connecting the movable platform and leg mechanism gives three constraints, thus the degrees of mobility of CaPaMan 2bis can be computed as n=(3*3+6)-3*3=3.

As shown in Fig. 1(b), if one refers to a single leg, the coordinates of the center of the k-th ball joint can be written as

$$x_{\nu} = h_{\nu} \cos \beta_{\nu} \tag{1}$$

$$y_k = b_k \cos \alpha_k \tag{2}$$

$$z_k = b_k \sin \alpha_k + h_k \sin \beta_k \tag{3}$$

Position and orientation of the platform can be obtained in closed-form through the expressions [6]

$$x = \frac{y_3 - y_2}{\sqrt{3}} - \frac{r_p}{2} (1 - \sin\phi) \cos(\psi - \theta)$$
 (4)

$$y = y_1 - r_n \left(\sin \psi \cos \theta + \cos \psi \sin \phi \sin \theta \right) \tag{5}$$

$$z = \frac{z_1 + z_2 + z_3}{3} \tag{6}$$

$$\theta = \sin^{-1} \left[2 \frac{y_1 + y_2 + y_3}{3r_{\rho} (1 + \sin \varphi)} \right] - \psi$$
 (7)

$$\psi = \tan^{-1} \left(\sqrt{3} \frac{z_3 - z_2}{2z_1 - z_2 - z_3} \right)$$
 (8)

$$\varphi = \cos^{-1}\left(\pm \frac{2}{3r_{\rho}} \sqrt{z_{1}^{2} + z_{2}^{2} + z_{3}^{2} - z_{1}z_{2} - z_{2}z_{3} - z_{1}z_{3}}\right) \text{ with}$$

$$\left(z \ge z_{1} \Rightarrow "+"; z < z_{1} \Rightarrow "-"\right) \tag{9}$$

where coordinates y_k and y_k are given by Equations (2) and (3). However, coordinates yk and z_k cannot be computed directly since the passive revolute

angles β_k . The solution of the direct kinematic problem for CaPaMan 2bis can be very complex and closed-form algebraic Equations (4)-(9) cannot be used as in CaPaMan [17].

3. Experimental setup

As explained above, regarding CaPaMan 2bis, it was very difficult to obtain a closed-form kinematic formulation. Thus, a suitable measuring system was used to measure the end-effector pose of CaPaMan 2 bis. Recently, a measuring system was developed at LARM, whose name is Milli-CaTraSys [16, 18, 19]. It is a cable-based architecture which can be used to measure the position and orientation of the end-effector of robotic systems. The Milli-CaTraSys design is based on a parallel manipulator configuration with 3-2-1 Gough-Stewart architecture. The kinematics of this type of measuring system has been solved in a closedform as a result of its special architecture by applying the trilateration technique. Moreover, Milli-CaTraSys can be used to apply a known wrench on the end-effector of a robotic system and, simultaneously, to measure corresponding end-effector displacements. Since it uses known masses m_i (with i=1,...,6) on the free ends of its cables, one can use Milli-CaTraSys for analyzing stiffness features of robotic systems. It has been already used successfully for evaluation of stiffness of robots as reported in [16]. Milli-CaTraSys consists of a fixed platform and a specific design end-effector with six cables. A set of six LVDT transducers is fixed at the end of each cable to determine the length change. The data acquisition and elaboration are executed by a NI DAQ data acquisition PCI 6024 card and a virtual instrument program in LABVIEW environment. A setup of the Milli-CaTraSys tracking system for measuring the motion of CaPaMan 2bis is shown in Figure 2. The six cables of Milli-CaTrasys are fixed on a specific designed end-effector that is installed on the movable platform MP of CaPaMan 2bis.

Three fixed points on the movable platform MP define a triangle on the end-effector.

A scheme of the experimental set-up for pose measuring of CaPaMa2 bis during a lab test at LARM is shown in Figure 3. In general, the experimental system consisted of two parts: the CaPaMan 2bis system and Milli-CaTraSys tracking system. The motion control system of prototype CaPaMan 2bis consists of a mechanical prototype as shown in Figure 1(a), whose leg mechanisms are actuated by Maxon RE25 DC motors, a control unit with the amplifiers of motors, and a NI data acquisition card. A program was developed in the LABVIEW environment to give control instructions and to handle the acquired data.

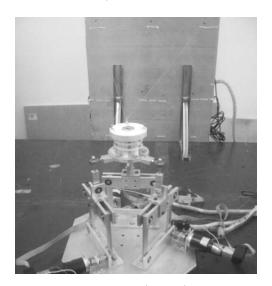


Figure 2. Experimental setup by using Milli-CaTraSys tracking system.

By measuring the length of the cables and using trilateration technique, the position and orientation of the Milli-CaTraSys end-effector can be determined in Cartesian space. This technique is based on suitable algebraic manipulations of distances I_i as function of the Cartesian coordinates x_H , y_H and z_H of a point H with respect to the fixed frame O-XYZ in the form

$$I_i^2 = (x_H - x_{Bi})^2 + (y_H - y_{Bi})^2 + (z_H - z_{Bi})^2$$
(with *i*=1,3,5) (10)

where X_{Bi} , Y_{Bi} and Z_{Bi} are the Cartesian coordinates of fixed points B_i at the Milli-CaTraSys base, as shown in Figure 4(a). After some algebraic manipulation, the position of H can be computed through its components as

$$x_{H} = H_{x} \left(-B_{2} - \sqrt{B_{2}^{2} - 4B_{1}B_{3}} \right) / 2B_{1} + E_{x}$$

$$y_{H} = H_{y} \left(-B_{2} - \sqrt{B_{2}^{2} - 4B_{1}B_{3}} \right) / 2B_{1} + E_{y}$$

$$z_{H} = \left(-B_{2} - \sqrt{B_{2}^{2} - 4B_{1}B_{3}} \right) / 2B_{1}$$
(11)

where

$$B_{1} = H_{x}^{2} + H_{y}^{2} + 1$$

$$B_{2} = 2H_{x} (E_{x} - X_{3}) + 2H_{y} (E_{y} - y_{3}) - 2Z_{3}$$

$$B_{3} = E_{x}^{2} + E_{y}^{2} + A_{3}^{2} - 2E_{x}X_{3} - 2E_{y}y_{3}B_{1}$$

$$H_{x} = -(Z_{21} + H_{y}Y_{21}) / X_{21}$$

$$E_{x} = (A_{1} + A_{2} - E_{y}Y_{21}) / X_{21}$$

$$E_{y} = \frac{-A_{1} + A_{2}}{Y_{32}X_{21}}$$
(12)

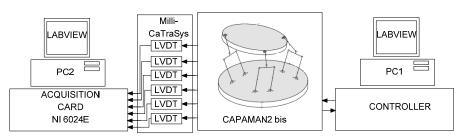


Figure 3. A scheme of the setup for the experimental tests of CaPaMan 2bis and Milli-CaTraSys at LARM.

with
$$(i, j=1,2,3), (i\neq i)$$

$$A_{i} = I_{i}^{2} - x_{i}^{2} - y_{i}^{2} - z_{i}^{2}$$

$$X_{ij} = x_{i} - x_{j}$$

$$Y_{ij} = y_{i} - y_{j}$$

$$Z_{ij} = z_{i} - z_{j}$$
(13)

Equations (11) and (12) express a general formulation of the trilateration scheme of Figure 4(a). Using expressions similar to Equations (11) to (13) for measured lengths I_2 and I_6 and HF distance, it is possible to determine the position of point F through its components x_F , y_F and z_F . Afterwards, the orientation of segment HF can be usefully described in the space through angles α , β and ψ , which can be obtained from the geometry of Figure 4(b) as

$$\alpha = \tan^{-1} \left(\frac{y_{H} - y_{F}}{x_{H} - x_{F}} \right)$$

$$\beta = \tan^{-1} \left(\frac{z_{H} - z_{F}}{\left(\left(y_{H} - y_{F} \right)^{2} + \left(x_{H} - x_{F} \right) \right)^{1/2}} \right)$$

$$\psi = \tan^{-1} \left(\frac{z_{Q} - z_{C}}{\sqrt{\left(y_{Q} - y_{C} \right)^{2} + \left(x_{Q} - x_{C} \right)^{2}}} \right)$$
(14)

Calibration work is needed both for CaPaMan 2bis and Milli-CaTraSys to have accuracy in control and measurement. A calibration process was carried out in order to determine a suitable initial configuration for the measuring system as well as to verify and to reduce the effects of systematic errors on its accuracy. During calibration, prototype CaPaMan 2bis is installed on a fixed rigid platform with base FP orthogonal with the platform of Milli-CaTraSys as shown in Figure 2. The position of point *O* of base FP can be easily measured with respect to an inertial reference frame fixed on the platform of Milli-CaTraSys.

The calibration and experimental tests of Milli-CaTraSys were carried out with LVDT B type transducers with a range of 100 mm. The calibration procedure can be summarized as in the flowchart of Figure 5, where ϵ_i and δ_i are acceptable calibration errors prescribed by the user. Depending on the required accuracy level, these values can be minimized even in an iterative process. Firstly, during the calibration procedure, each cable was placed in an initial reference position. A virtual instrument in the LabVIEW environment was developed for acquisition and processing of the data from the amplifiers of the LVDTs. The characteristic curve voltage-length of each LVDT can be defined by a line with its *Y*-axis

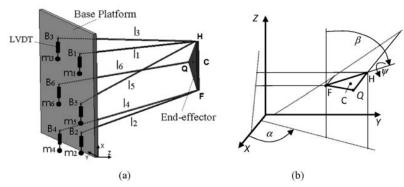


Figure 4. Schemes for Milli-CaTraSys system: (a) A 3D scheme of Milli-CaTraSys with the reference frame; (b) A scheme showing the end-effector orientation through set angles α , β and ψ .

intersection and scope. In the calibration procedure, these values are defined as the SUM Block value and the MULT Block value in the virtual instrument, respectively. Also, it is necessary to adjust the voltage offset as prescribed in the LVDT characteristic curve. Individual calibration of each LVDT was achieved by displacing the cables to know distances l_{Ri} . Calibration error δ_i is defined by the difference between measured distance l_{Mi} and known distance l_{Ri} .

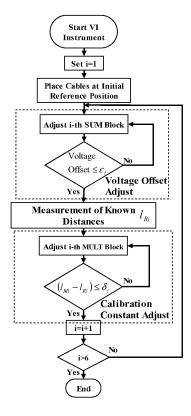


Figure 5. A flowchart of the calibration procedure for Milli-CaTraSys

4. Simulation model

In addition to experimental tests, in this work a simulation of CaPaMan 2bis and Milli-CaTraSys was carried out in a numerical way. Thus, a 3D model of CaPaMan 2bis and Milli-CaTraSys was built in

the ADAMS dynamic simulation environment. ADAMS is powerful software used successfully to simulate the dynamic behavior of vehicles [20, 21] and sometimes to design parallel kinematic systems [22]. By using this software, the kinematic and dynamic operation performances of CaPaMan 2bis was investigated through a proper model, as shown in Figure 6. The design parameters used for the 3D model of CaPaMan 2bis are those of the built prototype, and listed in Table I. Milli-CaTraSys cables were modeled by means of a set of six forces between the fixed points on the Milli-CaTraSys base and the attachment points on platform MP of CaPaMan 2bis. As shown in Figure 6, the total torque applied on the CaPaMan 2bis end-effector by the effect of these forces is modeled by means of a proper torque measured in the center of platform MP of CaPaMan 2bis.

Once the model of CaPaMan 2bis is properly adjusted in the ADAMS environment, different input motions were prescribed for the three input crank angles α_k (with k=1, 2, 3) to simulate different motions of the platform. Particularly, the same input motions were also used during the experimental tests. Table II lists the prescribed motions used for input crank angles αk during simulation.

As shown in Table II, three different input motions were prescribed for the simulation. For each case of study, the starting and end values of input crank angles α_k (with k=1, 2, 3), as well as actuation time t_k (with k=1, 2, 3) were given. Actually, since the three Maxon DC motors in the prototype of CaPaMan 2bis provide a position and velocity closed-loop algorithm control with a high dynamic performance, a cubic function was used to produce the input reference angles for each cycle as [9]

$$\alpha_k = c_0 + c_1 t + c_2 t^2 + c_3 t^3 \tag{15}$$

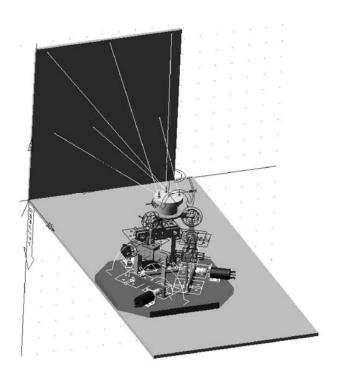


Figure 6. A 3D model of CaPaMan 2bis and Milli-CaTraSys built in ADAMS environment.

where four unknown constant values c_i (with i=0, 2, 3) in Equation (15) can be determined by solving four constraint equations. These are given by the values of position and velocity at the starting and end points. In each simulated case, position and orientation ofplatform MP was computed in the simulation environment by selecting proper reference frames and markers. Moreover, proper

ADAMS functions were built to measure correctly the position and orientation of CaPaMan 2bis MP platform in simulations. In particular, orientation functions were prescribed as in Equation (14). Additionally, the torque for each motor can also be determined to characterize the feasibility of the prescribed motion for the prototype of CaPaMan 2bis.

Motion:	Conditions									
	α_{1i}	α_{1f}	t_1	α_{2i}	α_{2f}	t_2	α_{3i}	α_{3f}	<i>t</i> ₃	
Three legs	78	102	4	78	102	4	78	102	4	
Two legs	80	95	4	100	85	4	90	90	4	
One Leg	78	108	4	90	90	4	90	90	4	

Table 2. Parameters for different input motions in simulation.

5. Analysis of simulation and experimental results

In this section, for each case of study listed in Table II, the position of end-effector center point *C* and orientation angles of the MP platform have been reported both for experiment results and computer simulation. In Figures (7)-(9), results are reported for the first case in Table II. The same motion was prescribed for the three legs of CaPaMan 2bis. The simulated motion starts from

point a_s , goes to b_s and ends at point c_s , as well as the measured motion starts from a_e , goes to b_e and ends at point c_e . In one cycle of motion, Figure 7(a), the differences between curves a_e - b_e and b_e - c_e could be due to the backlash and clearances in the mechanism of the prototype. As shown in Figure 7 (b), (c) and (d) the displacement is 2 mm in X axis, 1 mm in Y axis and 5.5 mm in Z axis, respectively. In Figure 8(a), three orientation angles ϕ , θ and ϕ are shown in 3D space.

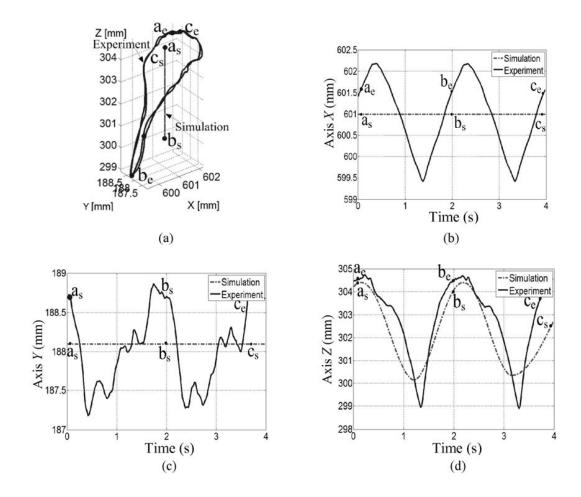


Figure 7. Position of platform MP for three-legged motion: (a) Position of point H in Cartesian space; (b) Position in the X axis; (c) Position in the Y axis; (d) Position in the Z axis

In Figures 8(b), (c) and (d), the orientation angles are plotted with respect to time. The maximum position difference is 2 mm as shown in Figure 9(a). The differences could be due to several

reasons, among them, errors in ADAMS modeling and computation, calibration errors in the experimental setup, and errors in the control algorithm in the CaPaMan 2bis system.

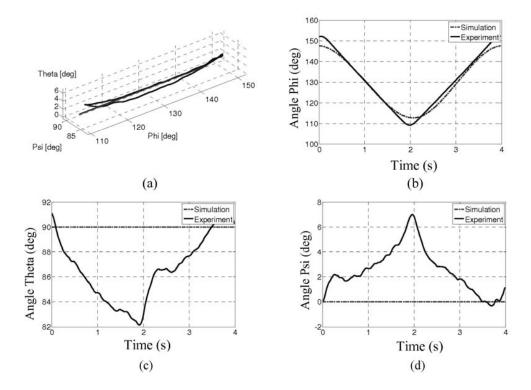


Figure 8. Orientation of platform MP for three-legged motion: (a) A view in 3D space; (b) Angle ϕ ; (c) Angle θ ; (d) Angle ψ .

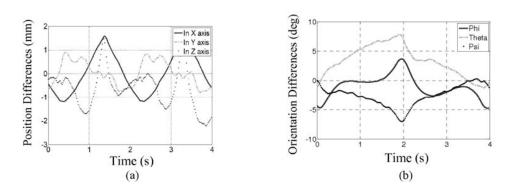


Figure 9. Differences between simulation and experimental results for three-legged motion: (a) Position differences; (b) Orientation angles differences

In four cycles of motion, Figures 10 (a) and (b) show the position and orientation repeatability for experiments with three legs of CaPaMan 2bis. The motion of the three legs of CaPaMan 2bis from -12 to 12 deg. provides as output a maximum motion for point C on the movable plate of about 5.5 mm in Z direction with a maximum rotation φ about the Z-axis of 42 deg. as shown in the plots of Figures 7(d) and 8(b), respectively. Table III summarizes the maximum movable ranges for the

three prescribed motions of CaPaMan 2bis described in Table II. For three-legged motion, the prototype performs a quasi translation motion along the Z axis with a rotation in angle ϕ . Therefore, it can be stated that the prototype of CaPaMan 2bis shows a good ability to run accurately operations with prescribed motion characteristics. From the above results, a more accurate operation can be expected by improving calibration and experimental setup.

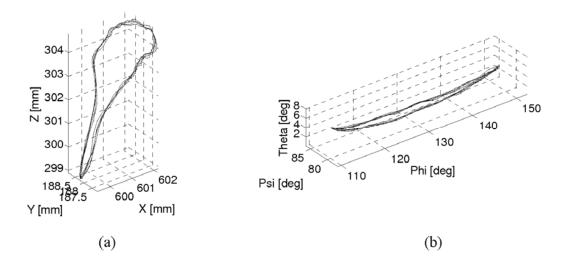


Figure 10. Experimental repeatability for four cycles of motion with three legs of CaPaMan 2bis: (a) Position of point H in Cartesian space; (b) Orientation of the MP platform in 3D space

	X_H [mm]		Y_H [mm]		Z_H [mm]		φ [deg]		θ [deg]		ψ [deg]	
	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max	Min	Max
Three legs	599.5	602.25	187.25	188.75	299	304.5	110	152	82	91	0	7
Two legs	580	610	175.80	179.50	297.6	304.3	128.5	132.4	89	92.7	-3	5
One leg	581	517	171	187	297.5	304	118	137.5	88.2	93.3	-4.2	3

Table 3. Measured maximum and minimum reaches for the three motions of CaPaMan 2bis

In Figures (11)-(13), results are reported for the second case of Table II. As shown in Figure 11, for the experiment results, the displacement is 30 mm in the X axis, 3.6 mm in the Y axis and 6.5 mm in the Z axis. The maximum position difference is 6 mm as shown in Figure 13(a). The three orientation angles are plotted in Figure 12.

Orientation angles φ , θ and ψ are near 130, 90 and 0 degrees, respectively. The maximum orientation difference is 4.8 degrees as shown in Figure 13(b). In four cycles of motion, Figures 14 (a) and (b) show the position and orientation repeatability for experiments with two legs of CaPaMan 2bis. Therefore, a quasi translation motion along the X axis has been obtained when two legs move.

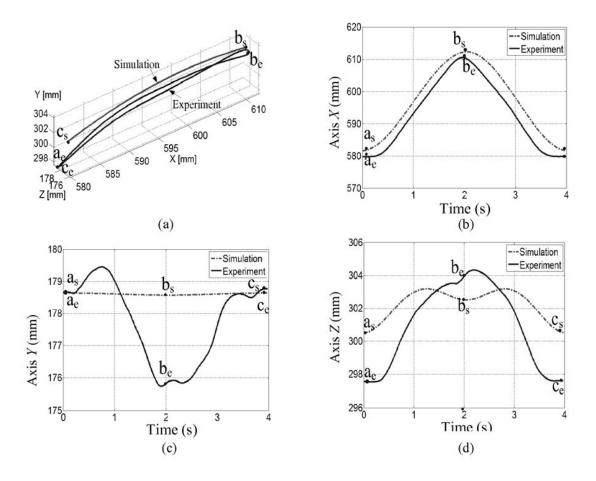


Figure 11. Position of platform MP for two-legged motion: (a) Position of point H in Cartesian space; (b) Position in the X axis; (c) Position in the Y axis; (d) Position in the Z axis

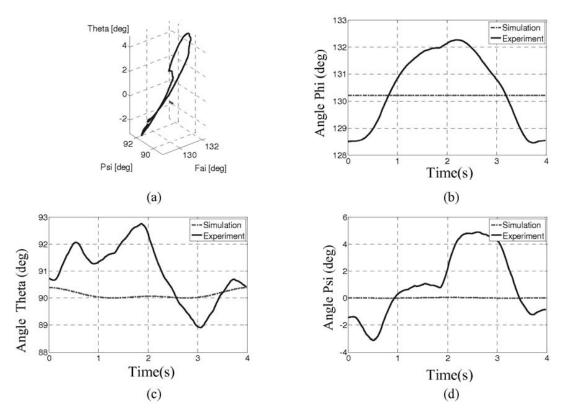


Figure 12. Orientation of platform MP for two-legged motion: (a) A view in 3D space; (b) Angle ϕ ; (c) Angle θ ; (d) Angle ψ .

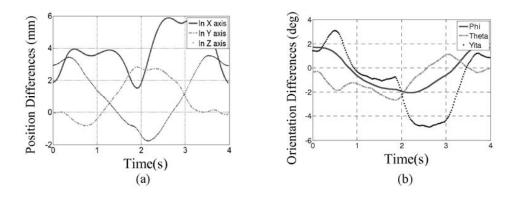


Figure 13. Differences between simulation and experimental results for two-legged motion: (a) Position differences; (b) Orientation angles differences

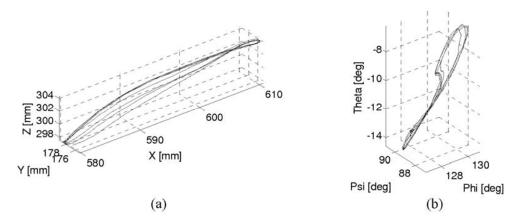


Figure 14. Experimental repeatability for four cycles of motion with two legs of CaPaMan 2bis: (a) Position of point H in Cartesian space; (b) Orientation of the MP platform in 3D space.

In Figures (15)-(17), results are reported for the third case of Table II. As shown in Figure 15(a), computed and measured trajectories of the endeffector point C are shown in a Cartesian space. Corresponding trajectories inthe X, Y and Z axis are shown in Figures 15(b), (c) and (d). Figure 16 shows

the orientation angles. The maximum position difference between computed and experimental results is ±6 mm as shown in Figure 17(a). In four cycles of motion, Figures 18 (a) and (b) show the position and orientation repeatability for experiments with one leg of CaPaMan 2bis.

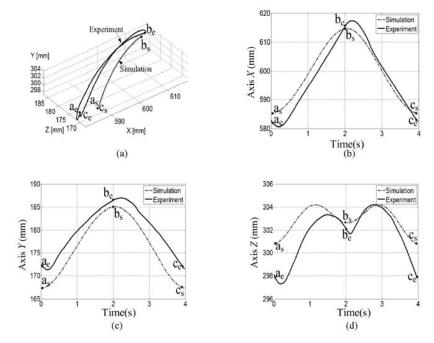


Figure 15. Position of the platform MP for one-legged motion: (a) Position of point H in Cartesian space; (b) Position in the X axis; (c) Position in the Y axis; (d) Position in the Z axis.

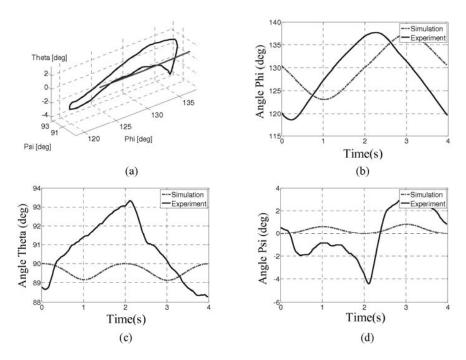


Figure 16. Orientation of platform MP for one-legged motion: (a) A view in 3D space; (b) Angle ϕ ; (c) Angle θ ; (d) Angle ψ .

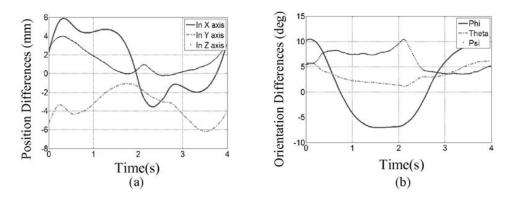


Figure 17. Differences between simulation and experimental results for one-legged motion: (a) Position differences; (b) Orientation angles differences

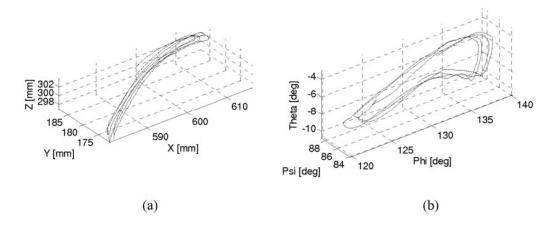


Figure 18. Experimental repeatability for four cycles of motion with one leg of CaPaMan 2bis: (a)

Position of point H in Cartesian space; (b) Orientation of the MP platform in 3D space

6. Conclusions

A characterization on the CaPaMan 2bis operation was presented based on experimental tests and computer simulations. Several cases of study were considered to evaluate the operation performance of prototype CaPaMan 2bis for different motion conditions. Feasible position and orientation motions were determined for different operation modes. Kinematical design and operation of the prototype was characterized bγ performance with smooth continuous actions and good repeatability ability. Comparison results show that different prescribed motions can be properly performed by the built prototype. In addition, the virtual model built in ADAMS environment can properly simulate the motion behavior of CaPaMan 2bis. Therefore, the formulated ADAMS model could be used as a motion simulator for CaPaMan 2bis for future studies of enhancements or novel applications of the parallel architecture of CaPaMan 2bis.

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